

# Problems and partial results in energy shaping

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24/03/2005



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## System definitions

- We will consider various flavours of systems, depending on whether they represent open-loop or closed-loop, or linear or nonlinear.

## Nonlinear system definitions

**Definition 1** A *simple mechanical control system* is a quadruple  $(Q, G, V, \mathcal{F} = \{F^1, \dots, F^m\})$  where

- (i)  $Q$  is an  $n$ -dimensional manifold,
- (ii)  $G$  is a Riemannian metric on  $Q$ ,
- (iii)  $V$  is a function on  $Q$ , and
- (iv)  $F^1, \dots, F^m$  are one-forms on  $Q$ , generating a subbundle of  $T^*Q$  which we denote by  $\mathcal{F}$ .

We assume all data to be at least of class  $C^\infty$ . •

- This will typically be the open-loop control system.

- The closed-loop system will be the following.

**Definition 2** A *forced simple mechanical system* is a quadruple  $(Q, G, V, F)$  where

- (i)  $Q$  is an  $n$ -dimensional manifold,
- (ii)  $G$  is a Riemannian metric on  $Q$ ,
- (iii)  $V$  is a function on  $Q$ , and
- (iv)  $F: TQ \rightarrow T^*Q$  is a bundle map over  $\text{id}_Q$  called the *external force*,

where we assume that all data is at least class  $C^\infty$ .

An external force  $F$  is

- (v) *dissipative* if  $F(v_q) = -R^b(v_q)$ , where  $R$  is a symmetric positive-semidefinite  $(0, 2)$ -tensor field called a *Rayleigh dissipation tensor*, is
- (vi) *linearly gyroscopic* if  $F(v_q) = -C^b(v_q)$ , where  $C$  is a skew-symmetric  $(0, 2)$ -tensor field called the *linear gyroscopic tensor*, and is
- (vii) *quadratically gyroscopic* if  $F(v_q) = -B^b(v_q)$ , where  $B$  is a  $(0, 3)$ -tensor field, called the *quadratic gyroscopic tensor*, satisfying  $B(u_q, v_q, w_q) = -B(v_q, u_q, w_q)$ , for all  $u_q, v_q, w_q \in \mathbb{T}Q$ , and where

$$\langle B^b(v_q); u_q \rangle = B(u_q, v_q, v_q). \quad \bullet$$

### Equations of motion for nonlinear systems

- For a simple mechanical control system  $(Q, \mathbb{G}, V, \mathcal{F} = \{F^1, \dots, F^m\})$ , the governing equations are

$$\begin{aligned} \overset{\mathbb{G}}{\nabla}_{\gamma'(t)} \gamma'(t) &= -\mathbb{G}^\# \circ dV(\gamma(t)) + \sum_{a=1}^m u^a(t) \mathbb{G}^\# \circ F^a(\gamma(t)) \\ (\ddot{q} + M^{-1}(q)C(q, \dot{q})\dot{q}) &= -M^{-1}(q)dV(q) + M^{-1}(q)G(q)u, \end{aligned}$$

where  $\overset{\mathbb{G}}{\nabla}$  is the Levi-Civita connection associated with  $\mathbb{G}$ .

- For a forced simple mechanical system  $(Q, \mathbb{G}, V, F)$ , the governing equations are

$$\begin{aligned} \overset{\mathbb{G}}{\nabla}_{\gamma'(t)} \gamma'(t) &= -\mathbb{G}^\# \circ dV(\gamma(t)) + \mathbb{G}^\# \circ F(\gamma'(t)) \\ (\ddot{q} + M^{-1}(q)C(q, \dot{q})\dot{q}) &= -M^{-1}(q)dV(q) + M^{-1}(q)F(q, \dot{q}). \end{aligned}$$

### Linear system definitions

- The linear open-loop control systems will have the following form.

**Definition 3** A *linear mechanical control system* is a quadruple  $(V, M, K, F)$  where

- (i)  $V$  is a finite-dimensional  $\mathbb{R}$ -vector space,
- (ii)  $M$  is an inner product on  $V$ ,
- (iii)  $K$  is a symmetric  $(0, 2)$ -tensor on  $V$ , and
- (iv)  $F \in L(\mathbb{R}^m; V^*)$ .

- Linear closed-loop systems have the following form.

**Definition 4** A *forced linear mechanical system* is a quadruple  $(V, M, K, (F_1, F_2))$  where

- (i)  $V$  is a finite-dimensional  $\mathbb{R}$ -vector space,
- (ii)  $M$  is an inner product on  $V$ ,
- (iii)  $K$  is a symmetric  $(0, 2)$ -tensor on  $V$ , and
- (iv)  $F_1$  and  $F_2$  are linear maps from  $V$  to  $V^*$  defining the *external force*.

If  $F_1 = 0$  and  $F_2$  is symmetric and negative semidefinite, then the external force is *dissipative*, and if  $F_1 = 0$  and  $F_2$  is skew-symmetric, then the external force is *gyroscopic*.

## Equations of motion for linear systems

- For a linear mechanical control system  $(V, M, K, F)$ , the governing equations are

$$\ddot{x}(t) + M^\# \circ K^b(x(t)) = M^\# \circ F(u(t)).$$

- For a forced linear mechanical system  $(V, M, K, (F_1, F_2))$ , the governing equations are

$$\ddot{x}(t) + M^\# \circ K^b(x(t)) = M^\# \circ F_1(x(t)) + M^\# \circ F_2(\dot{x}(t)).$$

## Problem formulation

### Energy shaping feedback for nonlinear systems

- Define  $\Lambda_{cl} = \mathbf{G}_{ol}^b \circ \mathbf{G}_{cl}^\#$ .

**Definition 5** An *energy shaping feedback* for a simple mechanical

$\Sigma_{ol} = (Q, \mathbf{G}_{ol}, V_{ol}, \mathcal{F})$  with closed-loop system

$\Sigma_{cl} = (Q, \mathbf{G}_{cl}, V_{cl}, -R_{cl}^b - C_{cl}^b - B_{cl}^b)$  is given by  $F: TQ \rightarrow \mathcal{F}$  with

$F = -F_{kin} - F_{pot} - F_{diss} - F_{gyr}$ , where

- (i)  $F_{kin}: TQ \rightarrow \mathcal{F}$  has the property that

$$\mathbf{G}_{ol}^\# \circ F_{kin}(\gamma'(t)) = \overset{\mathbf{G}_{cl}}{\nabla}_{\gamma'(t)} \gamma'(t) + \mathbf{G}_{cl}^\# \circ B_{cl}^b(\gamma'(t)) - \overset{\mathbf{G}_{ol}}{\nabla}_{\gamma'(t)} \gamma'(t),$$

- (ii)  $F_{pot}: Q \rightarrow \mathcal{F}$  has the property that

$$F_{pot}(\gamma(t)) = \Lambda_{cl} \circ dV_{cl}(\gamma(t)) - dV_{ol}(\gamma(t)),$$

(iii)  $F_{\text{diss}}: \text{TQ} \rightarrow \mathcal{F}$  has the property that

$$F_{\text{diss}}(\gamma'(t)) = \Lambda_{\text{cl}} \circ R_{\text{cl}}^{\flat}(\gamma'(t)),$$

(iv)  $F_{\text{gyr}}: \text{TQ} \rightarrow \mathcal{F}$  has the property that

$$F_{\text{gyr}}(\gamma'(t)) = \Lambda_{\text{cl}} \circ C_{\text{cl}}^{\flat}(\gamma'(t)).$$

### Energy shaping feedback for linear systems

**Definition 6** Let  $\Sigma_{\text{ol}} = (\mathbb{V}, M_{\text{ol}}, K_{\text{ol}}, F)$  be a linear mechanical control system. A *linear energy shaping feedback* for  $\Sigma_{\text{ol}}$  is a linear map  $u: \mathbb{V} \oplus \mathbb{V} \rightarrow \mathbb{R}^m$  with the property that there exists  $M_{\text{cl}}, K_{\text{shp}}, R_{\text{shp}} \in \text{TS}^2(\mathbb{V})$  and  $C_{\text{shp}} \in \mathbb{T}\wedge^2(\mathbb{V})$  such that  $M_{\text{cl}}$  is an inner product and such that

$$F \circ u(x, v) = -\Lambda_{\text{cl}} \circ K_{\text{shp}}^{\flat}(x) - \Lambda_{\text{cl}} \circ R_{\text{shp}}^{\flat}(v) - \Lambda_{\text{cl}} \circ C_{\text{shp}}^{\flat}(v),$$

where  $\Lambda_{\text{cl}} = M_{\text{ol}}^{\flat} \circ M_{\text{cl}}^{\sharp}$ .

## The central problem of energy shaping

### Problems 1

1. For a given open-loop system, determine the set of closed-loop systems.
2. Given a certain property for the closed-loop system, does there exist an energy shaping feedback for which the closed-loop system has this property (e.g., stability)? •

### Linear energy shaping<sup>1</sup>

- Given: The open-loop linear mechanical control system  $\Sigma_{\text{ol}} = (V, M_{\text{ol}}, K_{\text{ol}}, F)$ .
- Assume that the pair  $(\underbrace{M_{\text{ol}}^{\sharp} \circ K_{\text{ol}}^{\flat}}_A, \underbrace{M_{\text{ol}}^{\sharp} \circ F}_B)$  is controllable. (If not, then restrict to the controllable subspace.)
- Define  $E_{\Sigma_{\text{ol}}}$  to be the collection of  $A \in L(V; V)$  satisfying
  1.  $A = \underbrace{M_{\text{ol}}^{\sharp} \circ K_{\text{ol}}^{\flat} + M_{\text{ol}}^{\sharp} \circ F \circ L}_{A+B \circ K}$  for some  $L \in L(V; \mathbb{R}^m)$ , and
  2.  $A$  is diagonalisable over  $\mathbb{R}$ .

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<sup>1</sup>Zenkov, MTNS'02

**Proposition 1** Let  $\Sigma = (\mathbf{V}, M_{\text{ol}}, K_{\text{ol}}, F)$  be a linear mechanical control system. Then, for  $M_{\text{cl}}, K_{\text{cl}} \in \text{TS}^2(\mathbf{V})$  with  $M_{\text{cl}}$  an inner product, the following are equivalent:

- (i) there exists a linear feedback  $u: \mathbf{V} \oplus \mathbf{V} \rightarrow \mathbb{R}^m$  of the form  $x \oplus v \mapsto -L(x)$  for which the dynamics of the closed-loop system are those of the forced linear mechanical system  $(\mathbf{V}, M_{\text{cl}}, K_{\text{cl}}, (0, 0))$ ;
- (ii)  $M_{\text{cl}}^{\sharp} \circ K_{\text{cl}}^{\flat} \in E_{\Sigma_{\text{ol}}}$ .

**Corollary 1** A controllable linear mechanical control system can be stabilised by linear energy shaping feedback.

- One can construct a multitude of explicit ways to “pull apart”  $M_{\text{cl}}^{\sharp} \circ K_{\text{cl}}^{\flat}$  to yield  $M_{\text{cl}}$  and  $K_{\text{cl}}$ .

## Some results on potential shaping

### The classical result<sup>1</sup>

- Given: The open-loop simple mechanical control system  $\Sigma_{\text{ol}} = (\mathbf{Q}, \mathbb{G}_{\text{ol}}, V_{\text{ol}}, \mathcal{F})$ .
- Let  $\mathcal{F}^{(\infty)}$  be the largest integrable codistribution contained in  $\mathcal{F}$ .
- Assume that  $\mathcal{F}$  and  $\mathcal{F}^{(\infty)}$  are regular.
- Let  $C^{\infty}(\mathbf{Q})_{\mathcal{F}}$  denote the set of functions  $f$  for which  $df \in \Gamma^{\infty}(\mathcal{F}^{(\infty)})$ .

**Proposition 2 (van der Schaft 1986)** The difference between the closed- and open-loop potentials lies in  $C^{\infty}(\mathbf{Q})_{\mathcal{F}}$ .

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<sup>1</sup>van der Schaft, *Nonlinear Anal. TMA*, **10**(10), 1021–1035, 1986



### Potential shaping after kinetic shaping

- Now suppose that we have done some kinetic shaping to arrive at a closed-loop kinetic energy metric  $\mathbb{G}_{\text{cl}}$ . Recall that  $\Lambda_{\text{cl}} = \mathbb{G}_{\text{ol}}^b \circ \mathbb{G}_{\text{cl}}^\sharp$ .
- Let  $\mathcal{F}_{\text{cl}} = \Lambda_{\text{cl}}(\mathcal{F})$  and let  $\mathcal{F}_{\text{cl}}^{(\infty)}$  be the largest integrable codistribution contained in  $\mathcal{F}_{\text{cl}}$ .
- Assume that  $\mathcal{F}_{\text{cl}}$  and  $\mathcal{F}_{\text{cl}}^{(\infty)}$  are regular.
- Define

$$\text{PS}(\mathbb{Q}) = \{V_{\text{cl}} \in C^\infty(\mathbb{Q}) \mid dV_{\text{cl}} - \Lambda_{\text{cl}} \circ dV_{\text{ol}} \in \Gamma^\infty(\mathcal{F}_{\text{cl}})\}$$

$$\text{and } L(\text{PS}(\mathbb{Q})) = C^\infty(\mathbb{Q})_{\mathcal{F}_{\text{cl}}}.$$

**Proposition 3** *PS(Q) is an affine subspace (possibly empty) of  $C^\infty(\mathbb{Q})$  modelled on the subspace  $L(\text{PS}(\mathbb{Q}))$ .*

### Interpretation

- Recall the situation with the linear equation  $Ax = b$ :
  1.  $b \notin \text{image}(A)$ : No solutions.
  2.  $b \in \text{image}(A)$ : Set of solutions is an affine subspace modelled on  $\ker(A)$ .
- In the classical potential shaping case,  $b = 0$  in the analogue.
- We do not yet understand conditions for the analogue of  $b \notin \text{image}(A)$  or  $b \in \text{image}(A)$ , i.e., we do not understand the integrability of the potential shaping p.d.e.
- Note that the affine subspace is modelled on  $C^\infty(\mathbb{Q})_{\mathcal{F}_{\text{cl}}}$ .
  1. This subspace might be trivial, even when the classical energy shaping subspace,  $C^\infty(\mathbb{Q})_{\mathcal{F}}$ , is not.
  2. If  $\text{codim}(\mathcal{F}) = 1$ , then  $\text{codim}(\mathcal{F}_{\text{cl}}) = 1$ , and so Frobenius's Theorem guarantees that  $C^\infty(\mathbb{Q})_{\mathcal{F}_{\text{cl}}}$  is not trivial.

## Setting up the potential shaping problem for integrability tests<sup>1</sup>

- Think of the exterior derivative as a map, denoted by  $d_1$ , from  $J^1(\mathcal{F}_{cl})$  to  $T\wedge^2(TQ)$ .
- Define

$$P_{PS}(V_{ol})_q = \{j^1 F_{cl}(q) \mid d_1(j^1 F_{cl}(q)) = -d(\Lambda_{cl} \circ dV_{ol})(q)\}.$$

**Proposition 4** *Suppose that the first cohomology group of  $Q$  is zero. Then a function  $V_{cl}$  is a possible closed-loop potential function if and only if  $dV_{cl} = F_{cl} + \Lambda_{cl} \circ dV_{ol}$  where  $F_{cl}$  is a section of  $\mathcal{F}_{cl}$  having the property that  $j^1 F_{cl}$  takes values in  $P_{PS}(V_{ol})$ .*

- This result puts the kinetic shaping problem in a form where the techniques of Spencer, Serre, Quillen, Goldschmidt, etc. are applicable.

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<sup>1</sup>Spencer, *Ann. Math.*, **76**(3), 306–398 and 399–445, 1962,  
 Quillen, PhD thesis, Harvard University, 1964,  
 Serre, Appendix to Guillemin/Sternberg, *Bull. Amer. Math. Soc. (N.S.)*,  
**70**, 16–47, 1964  
 Goldschmidt, *J. Differential Geom.*, **1**, 269–307, 1967

## Formulations of the kinetic shaping problem

### An affine connection formulation

- For a general affine connection  $\nabla$  and Riemannian metric  $\mathbb{G}$  with its Levi-Civita connection  $\overset{\mathbb{G}}{\nabla}$ , define a  $(0, 3)$ -tensor field  $D_{\nabla, \mathbb{G}}$  by

$$\mathbb{G}(\nabla_X Y, Z) = \mathbb{G}(\overset{\mathbb{G}}{\nabla}_X Y, Z) + D_{\nabla, \mathbb{G}}(Z, X, Y).$$

- For a  $(0, k)$ -tensor  $A$  on  $V$ , define a symmetric (resp. skew-symmetric)  $(0, k)$ -tensor  $\text{Sym}(A)$  (resp.  $\text{Alt}(A)$ ) by

$$\text{Sym}(A)(v_1, \dots, v_k) = \frac{1}{k!} \sum_{\sigma \in S_k} A(v_{\sigma(1)}, \dots, v_{\sigma(k)}),$$

$$\text{resp. } \text{Alt}(A)(v_1, \dots, v_k) = \frac{1}{k!} \sum_{\sigma \in S_k} (-1)^{\text{sgn}(\sigma)} A(v_{\sigma(1)}, \dots, v_{\sigma(k)}).$$

- Think of  $\text{Sym}$  and  $\text{Alt}$  as linear maps from  $T_k^0(V)$  to  $T_k^0(V)$ .

- A  $(0, 3)$ -tensor  $A$  is *gyroscopic* if  $A(u, v, w) = -A(v, u, w)$   
is *torsional* if  $A(u, v, w) = -A(u, w, v)$   
Gyr(V): gyroscopic tensors  
Tor(V): torsional tensors
- For a Riemannian metric  $\mathbb{G}$ , define  $\text{KE}_{\mathbb{G}}: \text{TQ} \rightarrow \mathbb{R}$  by  
 $\text{KE}_{\mathbb{G}}(v_q) = \frac{1}{2}\mathbb{G}(v_q, v_q)$ .
- An affine connection  $\nabla$  is  *$\mathbb{G}$ -energy-preserving* if  
 $\mathcal{L}_{\gamma''(t)}\text{KE}_{\mathbb{G}}(\gamma'(t)) = 0$  for every geodesic  $\gamma$  of  $\nabla$ .

**Proposition 5** *The following are equivalent:*

- (i)  $\nabla$  is  $\mathbb{G}$ -energy preserving;
- (ii)  $\nabla\mathbb{G} \in \Gamma^\infty(\ker(\text{Sym}))$ ;
- (iii)  $D_{\nabla, \mathbb{G}} \in \Gamma^\infty(\ker(\text{Sym}))$ ;
- (iv) there exists tensor fields  $\Omega_{\nabla, \mathbb{G}} \in \Gamma^\infty(\text{T}\wedge^3(\text{TQ}))$ ,  
 $B_{\nabla, \mathbb{G}} \in \Gamma^\infty((\text{Gyr}(\text{TQ}) \cap \ker(\text{Alt})))$ , and  
 $\hat{T}_{\nabla, \mathbb{G}} \in \Gamma^\infty((\text{Tor}(\text{TQ}) \cap \ker(\text{Alt})))$  such that

$$\begin{aligned} \mathbb{G}(\nabla_X Y, Z) &= \mathbb{G}(\overset{\mathbb{G}}{\nabla}_X Y, Z) + B_{\nabla, \mathbb{G}}(Z, X, Y) \\ &\quad + \hat{T}_{\nabla, \mathbb{G}}(Z, X, Y) + \Omega_{\nabla, \mathbb{G}}(Z, X, Y), \end{aligned}$$

for all  $X, Y, Z \in \Gamma^\infty(\text{TQ})$ .

- *Discussion:*

1. If  $B$  is a gyroscopic tensor field, then there exists a unique energy-preserving, torsion-free affine connection  $\nabla$  such that

$$\mathbf{G}(\nabla_X X, Y) = \mathbf{G}(\overset{\mathbf{G}}{\nabla}_X X, Y) + B(Y, X, X)$$

for all  $X, Y \in \Gamma^\infty(\text{TQ})$ .

Explicitly,  $\nabla$  is defined by

$$\mathbf{G}(\nabla_X Y, Z) = \mathbf{G}(\overset{\mathbf{G}}{\nabla}_X Y, Z) + B_{\nabla, \mathbf{G}}(Z, X, Y),$$

where  $B_{\nabla, \mathbf{G}} = B - \text{Alt}(B)$ .

2. Changes the kinetic energy/quadratic gyroscopic force determination into a purely affine connection problem:

Find a Riemannian metric  $\mathbf{G}_{\text{cl}}$  and a  $\mathbf{G}_{\text{cl}}$ -energy preserving connection  $\overset{\text{cl}}{\nabla}$  such that

$$\overset{\text{cl}}{\nabla}_{\gamma'(t)} \gamma'(t) - \overset{\mathbf{G}_{\text{ol}}}{\nabla}_{\gamma'(t)} \gamma'(t) \in \mathbf{G}_{\text{ol}}^\#(\mathcal{F}_{\gamma(t)}).$$

### Setting up the kinetic shaping problem for integrability tests<sup>1</sup>

- Define

$$\text{ES}(\mathbf{Q}) = \Sigma_2^+(\text{TQ}) \times (\text{Gyr}(\text{TQ}) \cap \ker(\text{Alt}))$$

and

$$P_{\text{KS}}(\mathbf{G}_{\text{ol}})_q = \{(j^1 \mathbf{G}(q), j^1 B(q)) \in J^1(\text{ES}(\mathbf{Q})) \mid (\text{LC}(j^1 \mathbf{G}(q)) - \text{LC}(j^1 \mathbf{G}_{\text{ol}}(q)) + \mathbf{G}^\# B) \in \mathbf{G}_{\text{ol}}^\#(\mathcal{F} \otimes \text{TS}^2(\text{TQ}))\}.$$

**Proposition 6** *A Riemannian metric  $\mathbf{G}_{\text{cl}}$  and a gyroscopic tensor field  $B_{\text{cl}}$  solve the kinetic energy/quadratic gyroscopic problem if and only if the 1-jet of the section  $q \mapsto (\mathbf{G}_{\text{cl}}(q), B_{\text{cl}}(q))$  takes values in  $P_{\text{KS}}(\mathbf{G}_{\text{ol}})$ .*

- This result puts the kinetic shaping problem in a form where the techniques of Spencer, Serre, Quillen, Goldschmidt, etc. are applicable.

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<sup>1</sup>Ibid

## Energy shaping and linearisation

### Problem statement

- Given: A simple mechanical control system  $\Sigma_{\text{nonlin}} = (\mathbb{Q}, \mathbf{G}_{\text{ol}}, V_{\text{ol}}, \mathcal{F})$  with  $q_0$  an equilibrium point and  $\mathcal{F}$  regular.
- Let  $\Sigma_{\text{lin}} = (\mathbb{T}_{q_0}\mathbb{Q}, \mathbf{G}_{\text{ol}}(q_0), \text{Hess } V_{\text{ol}}(q_0), F)$  be its linearisation at  $q_0$ .

**Problem 1** When can a linear energy shaping feedback for the linearisation be implemented on the full system, and the implementation is ensured to also be energy shaping? •

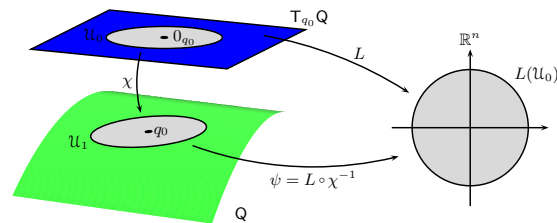
- What does “implemented” mean?
- If one is working in a particular set of coordinates, one “implements” without thinking about it.

- A coordinate-free version is the following.

**Definition 7** A *near identity diffeomorphism* at  $q_0 \in \mathbb{Q}$  is a triple  $(\chi, \mathcal{U}_0, \mathcal{U}_1)$ , where

- (i)  $\mathcal{U}_0 \subset \mathbb{T}_{q_0}\mathbb{Q}$  is a neighborhood of  $0_{q_0}$ ,
- (ii)  $\mathcal{U}_1 \subset \mathbb{Q}$  is a neighborhood of  $q_0$ , and
- (iii)  $\chi: \mathcal{U}_0 \rightarrow \mathcal{U}_1$  is a diffeomorphism satisfying
  - (a)  $\chi(0_{q_0}) = q_0$  and
  - (b)  $T_{0_{q_0}}\chi = \text{id}_{\mathbb{T}_{q_0}\mathbb{Q}}$  (where we make the natural identification of  $\mathbb{T}_{0_{q_0}}(\mathbb{T}_{q_0}\mathbb{Q})$  with  $\mathbb{T}_{q_0}\mathbb{Q}$ ). •

- Near identity diffeomorphisms have a simple relationship with coordinate charts:



- An *implementation* of a linear feedback  $u_{\text{lin}}: T_{q_0}Q \oplus T_{q_0}Q \rightarrow \mathbb{R}^m$  using a near identity diffeomorphism  $(\chi, \mathcal{U}_0, \mathcal{U}_1)$  is the control law  $u_{\text{nonlin}} = u_{\text{lin}} \circ T\chi^{-1}$ .

## Linearisation and potential shaping

**Proposition 7** For  $(Q, G_{\text{ol}}, V_{\text{ol}}, \mathcal{F})$  the following are equivalent:

- $\mathcal{F}$  is integrable;
  - there exists a family  $\tilde{\mathcal{F}}$  of input one-forms<sup>1</sup> and a near identity diffeomorphism such that every closed-loop potential of  $\Sigma_{\text{lin}}$  can be implemented as a potential shaping feedback for  $\Sigma_{\text{nonlin}}$ .
- **Punchline:** The obstructions to implementing a linearly shaped potential on the nonlinear system are the same as the obstructions to nonlinear potential shaping.

<sup>1</sup>Equivalent to  $\mathcal{F}$  in the sense that  $\tilde{\mathcal{F}} = \mathcal{F}$ .

## Linearisation and dissipative and gyroscopic forces

**Proposition 8** For  $(Q, G_{ol}, V_{ol}, \mathcal{F})$  the following are equivalent:

- (i)  $\mathcal{F}$  is integrable;
  - (ii) there exists a family  $\tilde{F}$  of input one-forms and a near identity diffeomorphism such that every closed-loop dissipative (resp. gyroscopic) force for  $\Sigma_{lin}$  can be implemented as a closed-loop dissipative (resp. gyroscopic) force for  $\Sigma_{nonlin}$ .
- **Punchline:** The obstructions implementing linear dissipative and gyroscopic forces on the nonlinear system are the same as the obstructions to nonlinear potential shaping.

## Interesting open question

**Question 1** When can a kinetic/potential shaping feedback for  $\Sigma_{lin}$  be implemented as a kinetic/potential shaping feedback for  $\Sigma_{nonlin}$ ?

## Open problems

1. Integrability of potential shaping p.d.e.
2. Integrability of kinetic shaping p.d.e.
3. Computable necessary or sufficient conditions for integrability.
4. What closed-loop kinetic energies allow useful potential shaping?
5. Complete the linearisation picture.
6. Closed-loop stability considerations.